

normalni napon i tangencijalni napon nosača izloženog savijanju

$$1. \quad \sigma = \frac{M_f \cdot R}{I_x} = \frac{M_t}{W_x} \leq \sigma_d$$

$$2. \quad \tau = \frac{M_f \cdot S_x'}{I_x \cdot \xi} = \frac{M_t}{W_x} \leq \tau_d$$

$$W_x = \frac{I_x}{y_{max}}$$

Zadatak 5.1.

Izračunati otporni moment pravougaonika odnosa strana $b/h = 1/2$.

$$I_x = \frac{b \cdot h^3}{12}; \quad I_y = \frac{h \cdot b^3}{12}; \quad \frac{b}{h} = \frac{1}{2} \rightarrow h = 2b; \quad b = \frac{h}{2}$$

$$W_x = \frac{b \cdot h^2}{6} = \frac{I_x}{\frac{h}{2}} = \frac{\frac{b \cdot h^3}{12}}{\frac{h}{2}} = \frac{\frac{h}{2} \cdot h^3}{12} = \frac{h^3}{12}$$

$$W_y = \frac{h \cdot b^2}{6} = \frac{I_y}{\frac{b}{2}} = \frac{\frac{h \cdot b^3}{12}}{\frac{b}{2}} = \frac{h \cdot (2b)^3}{12} = \frac{8 \cdot b^3}{12} = \frac{b^3}{3}$$

Zadatak 5.2.

Izračunati otporni moment kvadrata stranice $a = 3$ cm.

$$I_x = I_y = \frac{a^4}{12};$$

$$W_x = \frac{I_x}{\frac{a}{2}} = \frac{\frac{a^4}{12}}{\frac{a}{2}} = \frac{a^3}{6}$$

$$W_y = \frac{I_y}{\frac{a}{2}} = \frac{\frac{a^4}{12}}{\frac{a}{2}} = \frac{a^3}{6}$$

Zadatak 5.3.

Izračunati otporni moment kružnog prstena $\psi = 1/2$;

$$\psi = \frac{1}{2} = \frac{d}{D};$$

$$I_x = I_y = \frac{D^4 \cdot \pi}{64} (1 - \psi^4);$$

$$W_x = \frac{I_x}{\frac{D}{2}} = \frac{\frac{D^4 \cdot \pi}{64} (1 - \psi^4)}{\frac{D}{2}} = \frac{D^3 \cdot \pi}{32} (1 - \psi^4) = \frac{D^3 \cdot \pi}{32} \left(1 - \left(\frac{1}{2}\right)^4\right) = \frac{D^3 \cdot \pi}{32} \left(1 - \frac{1}{16}\right) = \frac{15}{16} \cdot \frac{D^3 \cdot \pi}{32}$$

$$W_x = \frac{15}{16} \cdot \frac{R^3 \cdot \pi}{4} = \frac{15}{64} \cdot R^3 \cdot \pi$$

Zadatak 5.4.

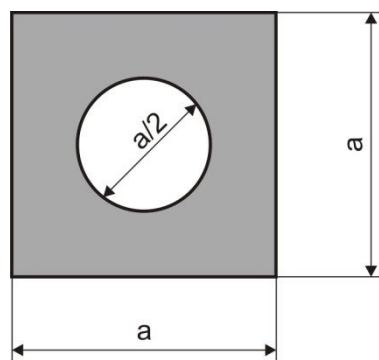
Iz kvadrata, stranice a, izvađena je centrična rupa d = ½ a. Izračunati otporni moment.

$$I_x^1 = I_y^1 = \frac{a^4}{12};$$

$$I_x^2 = I_y^2 = \frac{D^4 \cdot \pi}{64} = \frac{\left(\frac{a}{2}\right)^4 \cdot \pi}{64};$$

$$I_x = I_x^1 - I_x^2 = \frac{a^4}{12} - \frac{\left(\frac{a}{2}\right)^4 \cdot \pi}{64} = \frac{a^4}{4} \left(\frac{1}{3} - \frac{\pi}{256} \right)$$

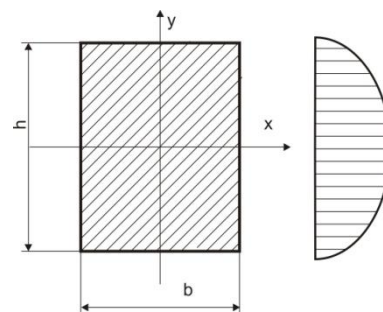
$$W_x = \frac{I_x}{\frac{a}{2}} = \frac{\frac{a^4}{4} \left(\frac{1}{3} - \frac{\pi}{256} \right)}{\frac{a}{2}} = \frac{a^3}{2} \left(\frac{1}{3} - \frac{\pi}{256} \right)$$

**Zadatak 5.5.**

Drvena pravougaona greda, preseka 6x8 cm prima tangencijalnu silu $F_T = 20$ kN. Koliki je najveći tangencijalni napon?

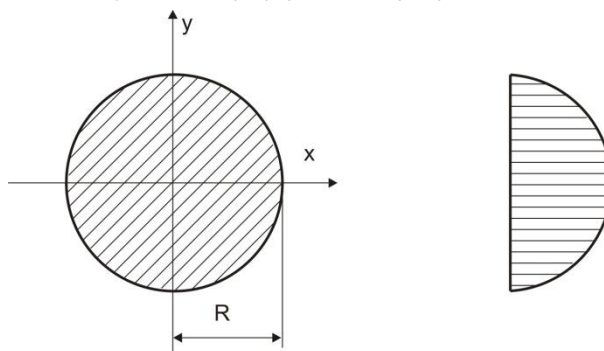
Najveći tangencijalni napon javlja se u vlaknima neutralne ose, jer je širina $b = \text{const}$. Obrazac za maksimalni tangencijalni napon, obrazac 104 u knjizi Otpornost materijala, dr. Inž. D. Rašković.

$$\tau_{max} = \frac{3}{2} \cdot \frac{F_{Tmax}}{A} = \frac{3}{2} \cdot \frac{20 \cdot 10^3}{8 \cdot 6 \cdot 10^{-4}} = 6\,250\,000 \text{ Pa} = 6.25 \text{ MPa}$$

**Zadatak 5.6.**

Greda kružnog preseka prima transversalnu silu $F_T = 22.25$ kN. U kom odnosu stoje tangencijalni naponi u vlaknima na neutralnoj osi i osi paralelnoj njoj na udaljenju od 1/2R ako je $R = 50$ mm?

Tangencijalni napon koji se javlja u vlaknima neutralne ose dat je izrazom za maksimalni tangencijalni napon, obrazac 105 u knjizi Otpornost materijala, dr. Inž. D. Rašković.



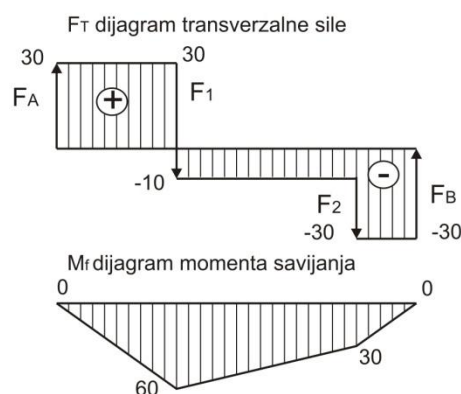
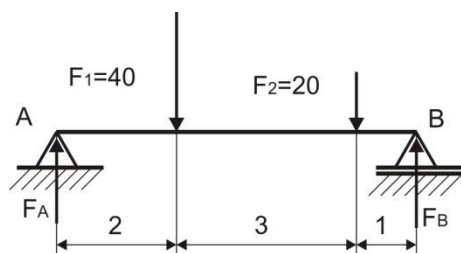
$$\tau_{max} = \frac{4}{3} \cdot \frac{F_{Tmax}}{A} = \frac{4}{3} \cdot \frac{22.25 \cdot 10^3}{\frac{5^2 \cdot \pi}{4} \cdot 10^{-4}} = 3\,828\,206 \text{ Pa} = 3.83 \text{ MPa}$$

Obrascem 105' definisan je napon u vlaknima paralelnim vlaknima neutralne ose

$$\tau = \tau_{max} \left[1 - \left(\frac{y}{R} \right)^2 \right] = 3\,828\,206 \left[1 - \left(\frac{0.5}{1} \right)^2 \right] = \frac{3}{4} \cdot 3\,828\,206 \text{ Pa} = 2.87 \text{ MPa}$$

Zadatak 5.7.

Prosta greda AB raspona $l = 6\text{m}$, opterećena je teretima $F_1=40\text{kN}$, $F_2=2\text{kN}$. Dimenzionisati čelilnu gredu, kružnog poprečnog preseka, ako je dozvoljeni napon $\sigma_d=120\text{MPa}$.



1. $\sum Z_i = 0$
2. $\sum Y_i = F_A - F_1 - F_2 + F_B = 0$
3. $\sum M_A = 2 \cdot F_1 + 5 \cdot F_2 - 6 \cdot F_B = 0$

$$F_B = \frac{2 \cdot F_1 + 5 \cdot F_2}{6} = \frac{2 \cdot 40 + 5 \cdot 20}{6} = 30 \text{ kN}$$

$$F_A = F_1 + F_2 - F_B = 40 + 20 - 30 = 30 \text{ kN}$$

Dimenzionisanje poprečnog preseka nosača

$$\sigma = \frac{M_f \cdot R}{I_x} = \frac{M_t}{W_x} \leq \sigma_d$$

$$W_x = \frac{d^3 \cdot \pi}{32}$$

$$d = \sqrt[3]{\frac{32 M_f}{\pi \sigma_d}} = \sqrt[3]{\frac{32 \cdot 60 \cdot 10^3}{\pi \cdot 120 \cdot 10^6}} = 0.172 \text{ m}$$

usvojeno standardno $d=0,18\text{ m}$

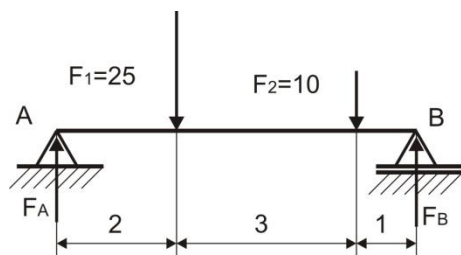
provera tangencijalnog napona

$$\tau_{max} = \frac{4}{3} \cdot \frac{F_{Tmax}}{A} = \frac{4}{2} \cdot \frac{4 \cdot 30 \cdot 10^3}{0.18^2 \cdot \pi \cdot 10^{-4}} = 1\,571\,900 \text{ Pa} = 1.57 \text{ MPa}$$

Zadatak 5.8.

Prosta greda AB raspona $l = 6\text{m}$, opterećena je teretima $F_1=25\text{kN}$, $F_2=10\text{kN}$. Dimenzionisati čelilnu gredu, pravougaonog poprečnog preseka odnosa $b/h=1/2$, ako je dozvoljeni napon $\sigma_d=120\text{MPa}$. Da li bi bio bolje iskorišćen I profil umesto pravougaonog profila?

1. $\sum Z_i = 0$
2. $\sum Y_i = F_A - F_1 - F_2 + F_B = 0$
3. $\sum M_A = 2 \cdot F_1 + 4 \cdot F_2 - 6 \cdot F_B = 0$

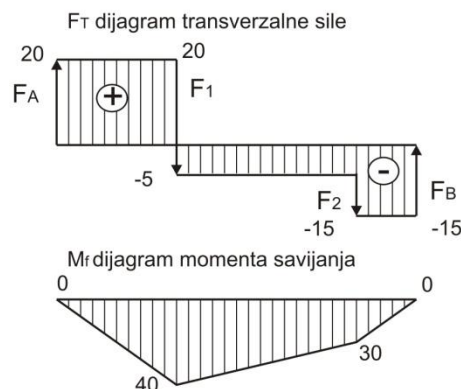


$$F_B = \frac{2 \cdot F_1 + 4 \cdot F_2}{6} = \frac{2 \cdot 25 + 4 \cdot 10}{6} = 15 \text{ kN}$$

$$F_A = F_1 + F_2 - F_B = 25 + 10 - 15 = 20 \text{ kN}$$

Maksimalni moment savijanja sa dijagrama

$$M_f = 40 \text{ kNm}$$



Dimenzionisanje poprečnog preseka nosača

$$\sigma = \frac{M_f \cdot R}{I_x} = \frac{M_t}{W_x} \leq \sigma_d \rightarrow W_x = \frac{M_t}{\sigma_d}$$

$$W_x = \frac{M_t}{\sigma_d} = \frac{40 \cdot 10^3}{120 \cdot 10^6} = 3.3 \cdot 10^{-4} \text{ m}^3$$

$$\frac{b}{h} = \frac{1}{2} \rightarrow h = 2b ; b = \frac{h}{2}$$

$$I_x = \frac{b \cdot h^3}{12} ; I_y = \frac{h \cdot b^3}{12} ;$$

$$W_x = \frac{b \cdot h^2}{12} = \frac{I_x}{\frac{h}{2}} = \frac{\frac{b \cdot h^3}{12}}{\frac{h}{2}} = \frac{\frac{h}{2} \cdot h^3}{12} = \frac{h^3}{12}$$

$$h = \sqrt[3]{\frac{12 \cdot M_t}{\sigma_d}} = \sqrt[3]{\frac{12 \cdot 40 \cdot 10^3}{120 \cdot 10^6}} = 0.1587 \text{ m}$$

standardni pravougaoni presek bxh = 0.16x0.08 m.

Težina utrošenog čelik za izradu pravougaone grede (za čelik $\gamma=7800 \text{ kg/m}^3$)

$$G_{prav} = V \cdot \gamma = l \cdot b \cdot h \cdot \gamma = 6 \cdot 0.16 \cdot 0.08 \cdot 7.8 \cdot 10^3 = 599 \text{ kg}$$

Iz tablica bira se I profil čiji je W_x jednak ili veći izračunatom otpornom momentu. Izračunato je $W_x = 330 \text{ cm}^3$, što uslovljava da se izabere IPN 24 čiji je $W_x = 354 \text{ cm}^3$ koji ima težinu po jedinici dužine $G' = 36.2 \text{ kg/m}$

$$G_{INP} = G' \cdot l = 6 \cdot 36.2 = 217.2 \text{ kg}$$

Ušteda je u korist INP standardnog valjanog profila

$$\Delta G = G_{prav} - G_{INP} = 599 - 217.2 = 381.84 \text{ kg}$$